

Penalty-Based Solution for the Interval Finite Element Methods

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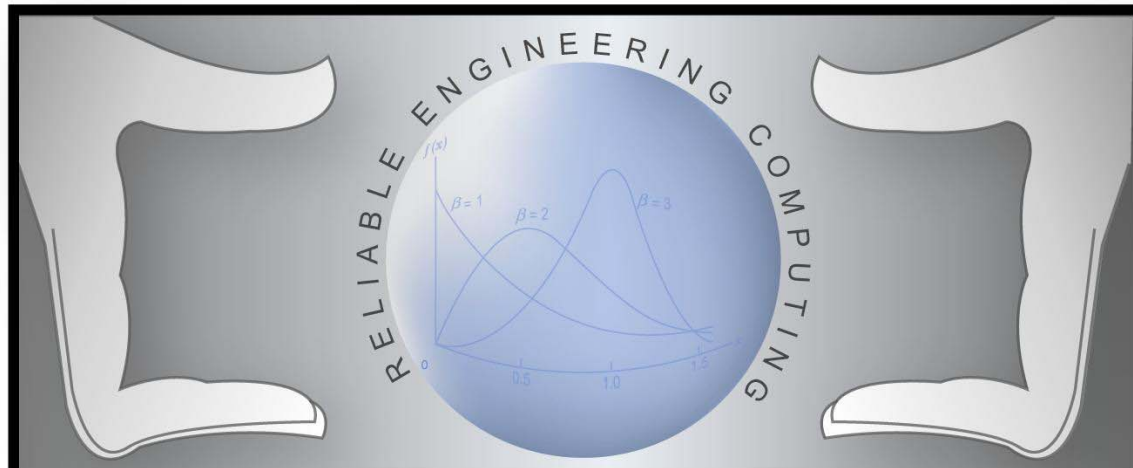
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Outline

- Interval Finite Elements
- Element-By-Element
- Penalty Approach
- Examples
- Conclusions

Center for Reliable Engineering Computing (REC)



Interval Calculator



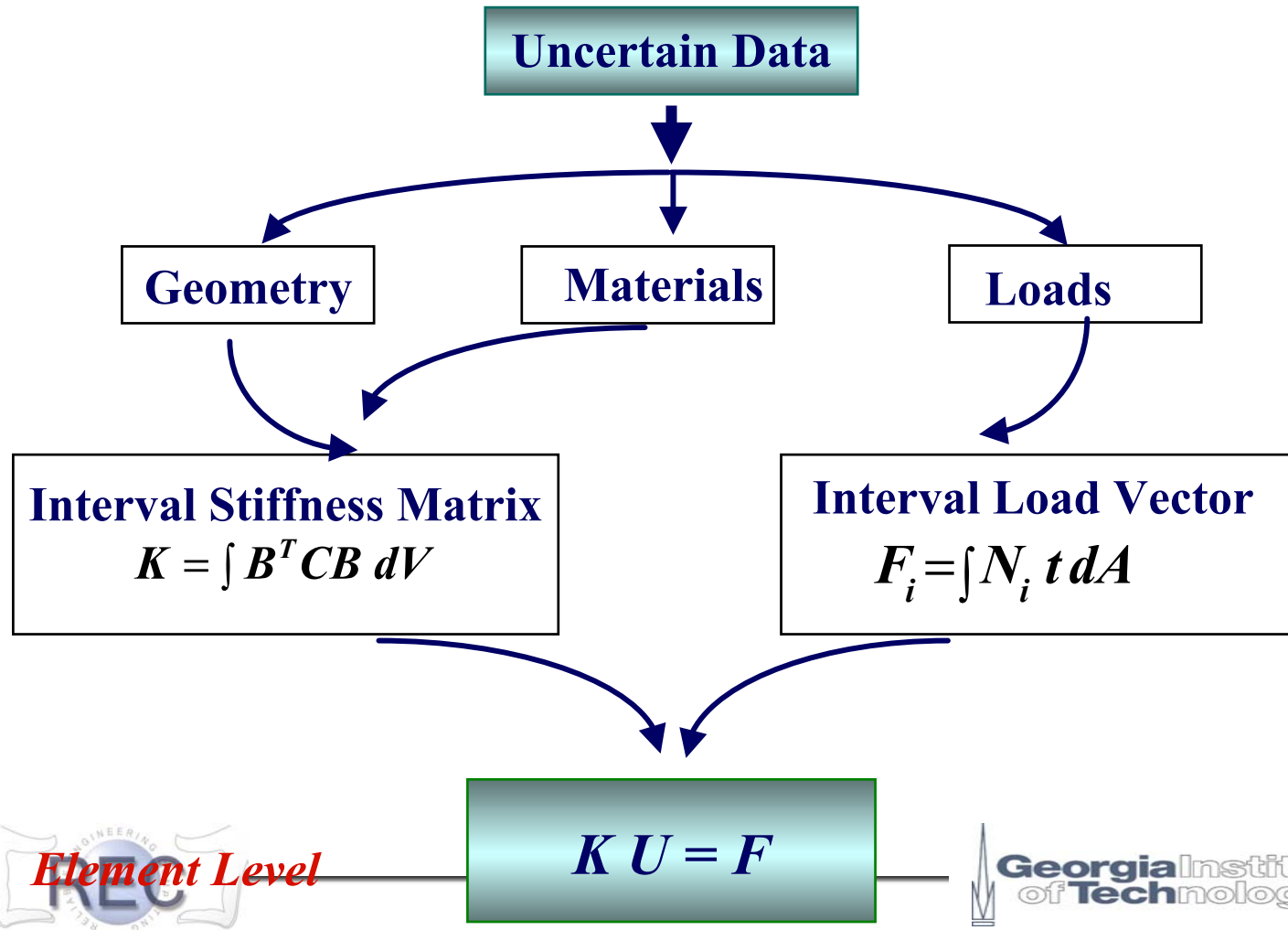
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Interval Finite Elements

- Follows conventional FEM
- Loads, nodal geometry and element materials are expressed as interval quantities
- Element-by-element method to avoid element stiffness coupling
- Lagrange Multiplier and Penalty function to impose compatibility
- Iterative approach to get enclosure
- Non-iterative approach to get exact hull for statically determinate structure

Interval Finite Elements



Interval Finite Elements

$$\mathbf{K} \mathbf{U} = \mathbf{F}$$

$\mathbf{K} = \int \mathbf{B}^T \mathbf{C} \mathbf{B} dV$ = Interval element stiffness matrix

$\mathbf{B}$ = Interval strain-displacement matrix

$\mathbf{C}$ = Interval elasticity matrix

$\mathbf{F} = [F_1, \dots, F_i, \dots, F_n]$ = Interval element load vector (traction)

$$F_i = \int N_i \mathbf{t} dA$$

N_i = Shape function corresponding to the i -th DOF

$\mathbf{t}$ = Surface traction

Finite Element

1. Load Dependency
2. Stiffness Dependency

Finite Element – Load Dependency

1. Load Dependency

$$\mathbf{P}_b = \sum_l \mathbf{L}^T \int \mathbf{N}^T \mathbf{b}(x) dx$$

The global load vector $\mathbf{P}_b$ can be written as

$$\mathbf{P}_b = \mathbf{M} \mathbf{q}$$

where $\mathbf{q}$ is the vector of interval coefficients of the load approximating polynomial

Finite Element – Load Dependency

Sharp solution for the interval displacement can be written as:

$$U = (K^{-1} \ M) q$$

Thus all non-interval values are multiplied first, the last multiplication involves the interval quantities

- *If this order is not maintained, the resulting interval solution will not be sharp*

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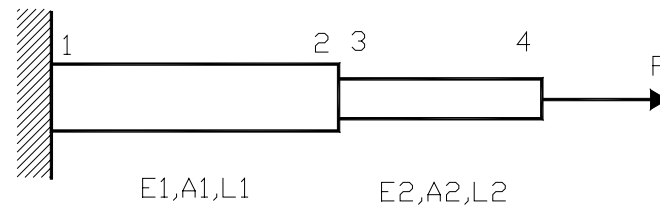
Finite Element – Element-by-Element Approach

➤ Stiffness Dependency

Coupling (assemblage process)

Finite Element – Element-by-Element Approach

➤ Coupling



$$k = \begin{pmatrix} k_1 + k_2 & -k_2 \\ -k_2 & k_2 \end{pmatrix}, \quad k^{-1} = \begin{pmatrix} \frac{1}{k_1} & \frac{1}{k_1} \\ \frac{1}{k_1} & \frac{k_1 + k_2}{k_1 k_2} \end{pmatrix}, \quad p = \begin{pmatrix} 0 \\ 1 \end{pmatrix}$$

$$u_2 = \frac{1}{k_1}, \quad u_3 = \frac{k_1 + k_2}{k_1 k_2} \quad (\text{over estimation in } u_3, r_3 = 3r_{3-\text{exact}})$$

$$u_2 = \frac{1}{k_1}, \quad u_3 = \frac{1}{k_1} + \frac{1}{k_2} \quad (\text{exact solution})$$

Finite Element – Element-by-Element Approach

- Element by Element to construct global stiffness
- Element level

$$K_1 = \begin{pmatrix} \frac{E_1 A_1}{L_1} & -\frac{E_1 A_1}{L_1} \\ -\frac{E_1 A_1}{L_1} & \frac{E_1 A_1}{L_1} \end{pmatrix} = \begin{pmatrix} E_1 & 0 \\ 0 & E_1 \end{pmatrix} \begin{pmatrix} \frac{A_1}{L_1} & -\frac{A_1}{L_1} \\ -\frac{A_1}{L_1} & \frac{A_1}{L_1} \end{pmatrix} = D_1 S_1 = S_1 D_1$$

Finite Element – Element-by-Element Approach

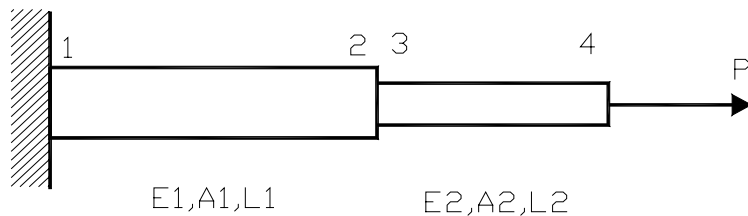
➤ K: block-diagonal matrix

$$K = \begin{pmatrix} K_1 & & \\ & K_2 & \\ & & \ddots \\ & & & K_n \end{pmatrix} = \begin{pmatrix} D_1 S_1 & & \\ & D_2 S_2 & \\ & & \ddots \\ & & & D_n S_n \end{pmatrix}$$

$$K = \begin{pmatrix} D_1 & & \\ & D_2 & \\ & & \ddots \\ & & & D_n \end{pmatrix} \begin{pmatrix} S_1 & & \\ & S_2 & \\ & & \ddots \\ & & & S_n \end{pmatrix} = \begin{pmatrix} S_1 & & \\ & S_2 & \\ & & \ddots \\ & & & S_n \end{pmatrix} \begin{pmatrix} D_1 & & \\ & D_2 & \\ & & \ddots \\ & & & D_n \end{pmatrix}$$

Finite Element – Element-by-Element Approach

➤ Element-by-Element



$$K = DS = SD$$

$$D = \begin{pmatrix} E_1 & 0 & 0 & 0 \\ 0 & E_1 & 0 & 0 \\ 0 & 0 & E_2 & 0 \\ 0 & 0 & 0 & E_2 \end{pmatrix}$$

$$S = \begin{pmatrix} \frac{A_1}{L_1} & -\frac{A_1}{L_1} & 0 & 0 \\ -\frac{A_1}{L_1} & \frac{A_1}{L_1} & 0 & 0 \\ 0 & 0 & \frac{A_2}{L_2} & -\frac{A_2}{L_2} \\ 0 & 0 & -\frac{A_2}{L_2} & \frac{A_2}{L_2} \end{pmatrix}$$

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- **Penalty Approach**
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Finite Element – Present Formulation

- In steady-state analysis-variational formulation

$$\Pi = \frac{1}{2} \mathbf{U}^T \mathbf{K} \mathbf{U} - \mathbf{U}^T \mathbf{P}$$

- With the constraints $t = \mathbf{C} \mathbf{U} = 0$

$$\mathbf{C} = \begin{pmatrix} 0 & 1 & -1 & 0 \end{pmatrix} \text{ and } \mathbf{U}^T = (\mathbf{U}_1 \quad \mathbf{U}_2 \quad \mathbf{U}_3 \quad \mathbf{U}_4)$$

- Adding the penalty function $\frac{1}{2} t^T \alpha t$

α : a diagonal matrix of penalty numbers

$$\Pi^* = \frac{1}{2} \mathbf{U}^T \mathbf{K} \mathbf{U} - \mathbf{U}^T \mathbf{P} + \frac{1}{2} t^T \alpha t$$

Finite Element – Present Formulation

➤ Invoking the stationarity of Π^* , that is $\delta\Pi^* = 0$

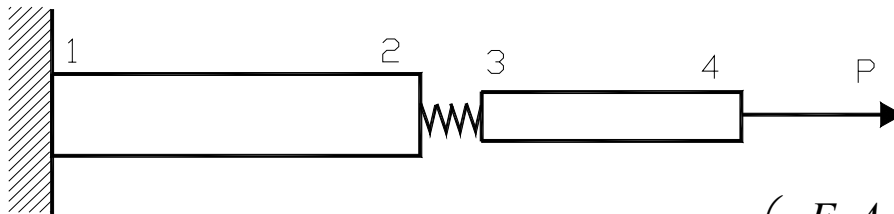
$$(K + C^T \alpha C)U = P$$

$$(K + Q)U = P$$

$$C^T = \begin{pmatrix} 0 \\ 1 \\ -1 \\ 0 \end{pmatrix} \quad Q = C^T \alpha C = \alpha \begin{pmatrix} 0 & 0 & 0 & 0 \\ 0 & 1 & -1 & 0 \\ 0 & -1 & 1 & 0 \\ 0 & 0 & 0 & 0 \end{pmatrix}$$

Finite Element – Penalty Approach

- The physical meaning of Q is an addition of a large spring stiffness



$$Q = \alpha \begin{pmatrix} 0 & 0 & 0 & 0 \\ 0 & 1 & -1 & 0 \\ 0 & -1 & 1 & 0 \\ 0 & 0 & 0 & 0 \end{pmatrix}$$

$$K + Q = \begin{pmatrix} \frac{E_1 A_1}{L_1} & -\frac{E_1 A_1}{L_1} & 0 & 0 \\ -\frac{E_1 A_1}{L_1} & \frac{E_1 A_1}{L_1} + \alpha & -\alpha & 0 \\ 0 & -\alpha & \frac{E_2 A_2}{L_2} + \alpha & -\frac{E_2 A_2}{L_2} \\ 0 & 0 & -\frac{E_2 A_2}{L_2} & \frac{E_2 A_2}{L_2} \end{pmatrix}$$

Finite Element – Penalty Approach

- Interval system of equations

$$(K + Q)U = P \quad \text{or} \quad AU = P$$

- where

$$A = \{\tilde{A} \in R^{n \times n} \mid \tilde{A}_{ik} \in A_{ik} \text{ for } i, k = 1, \dots, n\}$$

$$P = \{\tilde{P} \in R^{n \times 1} \mid \tilde{P}_i \in P_i \text{ for } i = 1, \dots, n\}$$

- and

$$D = \{\tilde{D} \in R^{n \times n} \mid \tilde{D}_{ii} \in D_{ii} \text{ for } i = 1, \dots, n\}$$

$$K = DS = SD$$

Finite Element – Penalty Approach

- The solution will have the following form

$$RP - (I - RA)U \subseteq \text{int}(U)$$

- where $R = \text{inverse mid } (A)$ and $U = U^* + U_0$

- or

$$RP - RAU_0 + (I - RA)U^* \subseteq \text{int}(U^*)$$

$$z + CU^* \subseteq \text{int}(U^*)$$

Finite Element – Penalty Approach

$$z = RP - RAU_0 = RP - R(K + Q)U_0$$

$$z = RP - RQU_0 - RSDU_0 = RP - RQU_0 - RSM\delta$$

➤ $R = (S + Q)^{-1}$ and $U_0 = RP$

$$C = I - RA = I - RK - RQ = I - RQ - RSD$$

➤ Algorithm converges if and only if $\rho(|C|) < 1$

Finite Element – Penalty Approach

➤ Rewrite $DU_0 = M\delta$

$$\begin{pmatrix} E_1 & 0 & 0 & 0 \\ 0 & E_1 & 0 & 0 \\ 0 & 0 & E_2 & 0 \\ 0 & 0 & 0 & E_2 \end{pmatrix} \begin{pmatrix} U_{01} \\ U_{02} \\ U_{03} \\ U_{04} \end{pmatrix} = \begin{pmatrix} U_{01} & 0 \\ U_{02} & 0 \\ 0 & U_{03} \\ 0 & U_{04} \end{pmatrix} \begin{pmatrix} E_1 \\ E_2 \end{pmatrix} = \begin{pmatrix} E_1 U_{01} \\ E_1 U_{02} \\ E_2 U_{03} \\ E_2 U_{04} \end{pmatrix}$$

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Examples

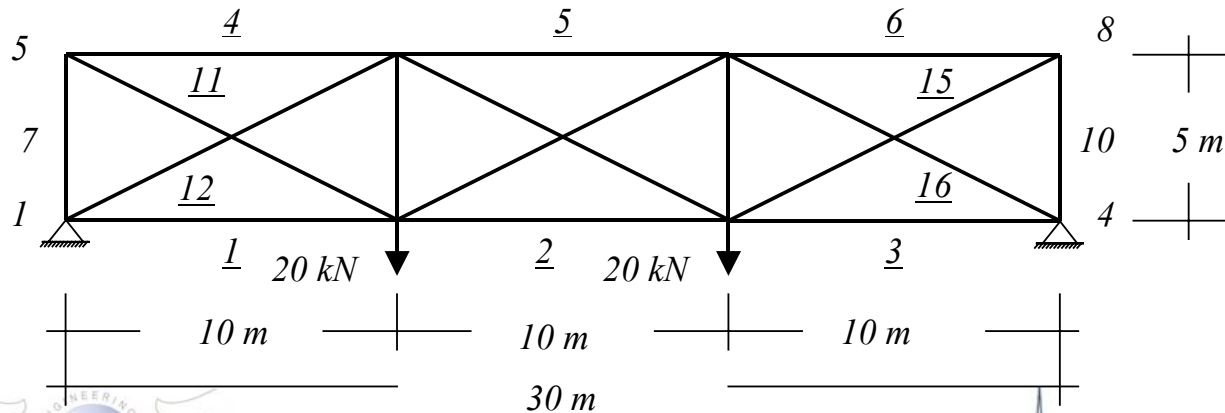
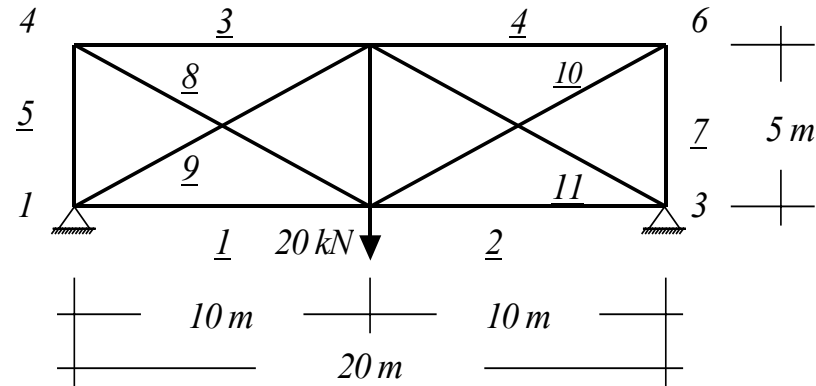
- Statically indeterminate (general case)
 - Two-bay truss
 - Three-bay truss
 - Four-bay truss
 - Statically indeterminate beam
- Statically determinate
 - Three-step bar

Examples – Stiffness Uncertainty

- Two-bay truss
- Three-bay truss

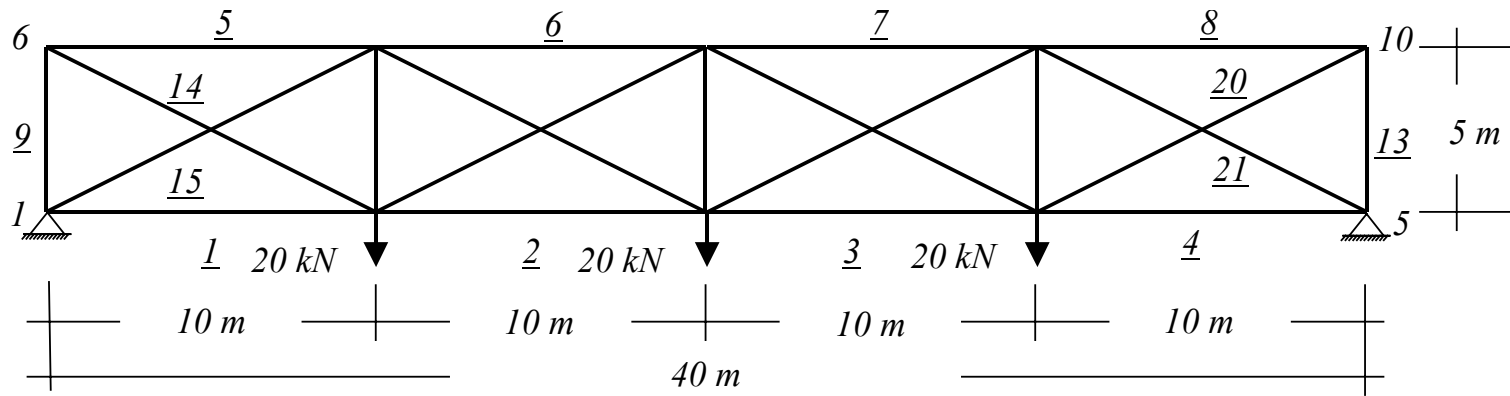
$$A = 0.01 \text{ m}^2$$

$$E \text{ (nominal)} = 200 \text{ GPa}$$



Examples – Stiffness Uncertainty

➤ Four-bay truss



Examples – Stiffness Uncertainty 1%

➤ Two-bay truss

Two bay truss (11 elements) with 1% uncertainty in Modulus of Elasticity, $E = [199, 201]$ GPa

	V2(LB)(m)	V2(UB)(m)	U4(LB)(m)	U4(UB)(m)
Comb $\times 10^{-4}$	– 2.00326	– 1.98333	0.38978	0.40041
Present $\times 10^{-4}$	– 2.00338	– 1.98302	0.38965	0.40050
error	– 0.006%	0.015%	0.033%	– 0.023%

Examples – Stiffness Uncertainty 1%

➤ Three-bay truss

Three bay truss (16 elements) with 1% uncertainty in Modulus of Elasticity, $E = [199, 201]$ GPa

	V2(LB)(m)	V2(UB)(m)	U5(LB)(m)	U5(UB)(m)
Comb $\times 10^{-4}$	– 5.84628	– 5.78663	1.54129	1.56726
Present $\times 10^{-4}$	– 5.84694	– 5.78542	1.5409	1.5675
error	– 0.011%	0.021%	0.025%	– 0.015%

Examples – Stiffness Uncertainty 1%

➤ Four-bay truss

Four-bay truss (21 elements) with 1% uncertainty in Modulus of Elasticity, $E = [199, 201]$ GPa

	V2(LB)(m)	V2(UB)(m)	U6(LB)(m)	U6(UB)(m)	V6(LB)(m)	V6(UB)(m)
Comb $\times 10^{-4}$	- 17.7729	- 17.5942	3.83417	3.88972	- 0.226165	- 0.220082
Present $\times 10^{-4}$	- 17.7752	- 17.5902	3.83268	3.89085	- 0.226255	- 0.21995
error	- 0.013%	0.023%	0.039%	- 0.029%	- 0.040%	0.060%

Examples – Stiffness Uncertainty 5%

➤ Two-bay truss

Two bay truss (11 elements) with 5% uncertainty in Modulus of Elasticity, $E = [195, 205]$ GPa

	V2(LB)(m)	V2(UB)(m)	U4(LB)(m)	U4(UB)(m)
Comb $\times 10^{-4}$	- 2.04435	- 1.94463	0.36866	0.42188
Present $\times 10^{-4}$	- 2.04761	- 1.93640	0.36520	0.42448
error	- 0.159%	0.423%	0.939%	- 0.616%

Examples – Stiffness Uncertainty 5%

➤ Three-bay truss

Three bay truss (16 elements) with 5% uncertainty in Modulus of Elasticity, $E = [195, 205]$ GPa

	V2(LB)(m)	V2(UB)(m)	U5(LB)(m)	U5(UB)(m)
Comb $\times 10^{-4}$	– 5.9692233	– 5.6708065	1.4906613	1.6195115
Present $\times 10^{-4}$	– 5.98838	– 5.63699	1.47675	1.62978
error	– 0.321%	0.596%	0.933%	– 0.634%

Examples – Stiffness Uncertainty 10%

➤ Two-bay truss

Two bay truss (11 elements) with 10% uncertainty in Modulus of Elasticity, $E = [190, 210]$ GPa

	V2(LB)(m)	V2(UB)(m)	U4(LB)(m)	U4(UB)(m)
Comb $\times 10^{-4}$	– 2.09815	– 1.89833	0.34248	0.44917
Present $\times 10^{-4}$	– 2.11418	– 1.86233	0.32704	0.46116
error	– 0.764%	1.896%	4.508%	– 2.669%

Examples – Stiffness Uncertainty 10%

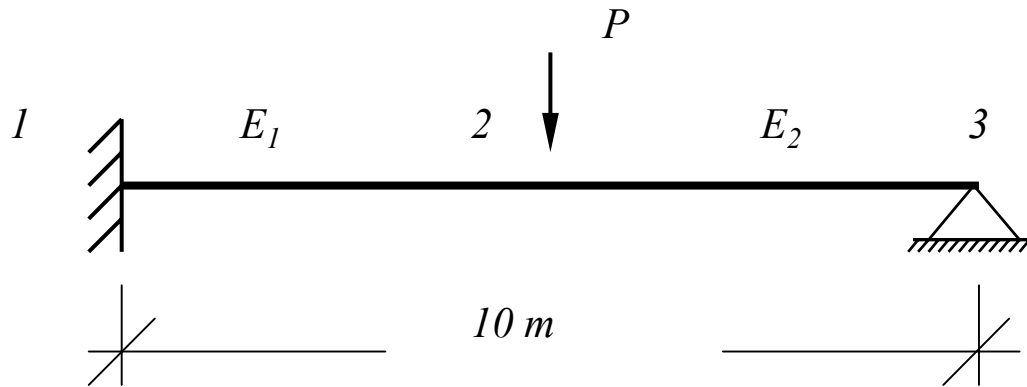
➤ Three-bay truss

Three bay truss (16 elements) with 10% uncertainty in Modulus of Elasticity, $E = [190, 210]$ GPa

	V2(LB)(m)	V2(UB)(m)	U5(LB)(m)	U5(UB)(m)
Comb $\times 10^{-4}$	– 6.13014	– 5.53218	1.42856	1.68687
Present $\times 10^{-4}$	– 6.22965	– 5.37385	1.36236	1.7383
error	– 1.623%	2.862%	4.634%	– 3.049%

Examples – Stiffness and Load Uncertainty

➤ Statically indeterminate beam



$$A = 0.086 \text{ m}^2 \quad I = 10^{-4} \text{ m}^4 \quad E \text{ (nominal)} = 200 \text{ GPa}$$

Examples – Stiffness and Load Uncertainty

➤ Statically indeterminate beam

Statically indeterminate beam (2 elements) with 1% uncertainty in Modulus of Elasticity, $E = [199, 201]$ GPa, 10% uncertainty in Load, $P=[9.5, 10.5]$ kN

	V2(LB)(m)	V2(UB)(m)	θ^2 (LB)(rad)	θ^2 (UB)(rad)
Comb $\times 10^{-3}$	– 4.80902	– 4.307888	1.47699	1.648869
Present $\times 10^{-3}$	– 4.80949	– 4.30487	1.47565	1.64928
error	– 0.00977%	0.07006%	0.09073%	– 0.02493%

Examples – Stiffness and Load Uncertainty

➤ Statically indeterminate beam

Statically indeterminate beam (2 elements) with 1% uncertainty in Modulus of Elasticity, $E = [199, 201]$ GPa, 20% uncertainty in Load, $P=[9, 11]$ kN

	V2(LB)(m)	V2(UB)(m)	θ_2 (LB)(rad)	θ_2 (UB)(rad)
Comb $\times 10^{-3}$	– 5.03821	– 4.081157	1.399254	1.727387
Present $\times 10^{-3}$	– 5.03884	– 4.07552	1.39672	1.7282
error	– 0.01250%	0.13812%	0.18110%	– 0.04707%

Examples – Stiffness and Load Uncertainty

➤ Statically indeterminate beam

Statically indeterminate beam (2 elements) with 1% uncertainty in Modulus of Elasticity, $E = [199, 201]$ GPa, 40% uncertainty in Load, $P=[8, 12]$ kN

	V2(LB)(m)	V2(UB)(m)	θ_2 (LB)(rad)	θ_2 (UB)(rad)
Comb $\times 10^{-3}$	– 5.49623	– 3.62769	1.234378	1.8844221
Present $\times 10^{-3}$	– 5.49751	– 3.61684	1.23888	1.88604
error	– 0.02329%	0.29909%	– 0.36472%	– 0.08586%

Examples – Stiffness and Load Uncertainty

➤ Three-bay truss

Three bay truss (16 elements) with 1% uncertainty in Modulus of Elasticity, $E = [199, 201]$ GPa, 5% uncertainty in Load, $P = [19.5, 20.5]$ kN

	V2(LB)(m)	V2(UB)(m)	U4(LB)(m)	U4(UB)(m)
Comb $\times 10^{-4}$	– 2.05334	– 1.93374	0.38003	0.41042
Present $\times 10^{-4}$	– 2.05381	– 1.93259	0.37953	0.41062
error	– 0.023%	0.060%	0.132%	– 0.050%

Examples – Stiffness and Load Uncertainty

➤ Three-bay truss

Three bay truss (16 elements) with 1% uncertainty in Modulus of Elasticity, $E = [199, 201]$ GPa, 10% uncertainty in Load, $P = [19, 21]$ kN

	V2(LB)(m)	V2(UB)(m)	U4(LB)(m)	U4(UB)(m)
Comb $\times 10^{-4}$	– 2.10342	– 1.88416	0.37029	0.42043
Present $\times 10^{-4}$	– 2.10425	– 1.88215	0.36941	0.42074
error	– 0.039%	0.107%	0.237%	– 0.075%

Examples – Stiffness and Load Uncertainty

➤ Three-bay truss

Three bay truss (16 elements) with 1% uncertainty in Modulus of Elasticity, $E = [199, 201]$ GPa, 20% uncertainty in Load, $P = [18, 22]$ kN

	V2(LB)(m)	V2(UB)(m)	U4(LB)(m)	U4(UB)(m)
Comb $\times 10^{-4}$	– 2.20359	– 1.78499	0.35080	0.44045
Present $\times 10^{-4}$	– 2.20511	– 1.78129	0.34917	0.44098
error	– 0.069%	0.207%	0.465%	– 0.121%

Examples – Statically determinate

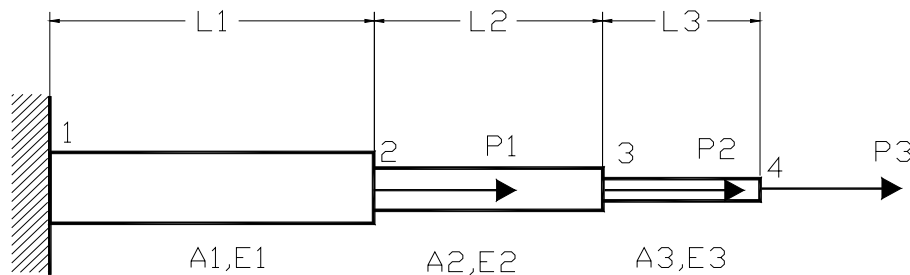
➤ Three-step bar

$E1 = [18.5, 21.5]\text{MPa}$ (15% uncertainty)

$E2 = [21.875, 28.125]\text{MPa}$ (25% uncertainty)

$E3 = [24, 36]\text{MPa}$ (40% uncertainty)

$P1 = [-9, 9]\text{kN}$ $P2 = [-15, 15]\text{kN}$ $P3 = [2, 18]\text{kN}$



Examples – Statically determinate

➤ Statically determinate 3-step bar

	U1(LB)(m)	U1(UB)(m)	U2(LB)(m)	U2(UB)(m)	U3(LB)(m)	U3(UB)(m)
Comb $\times 10^{-3}$	- 4.756756	9.081081	- 7.72818	16.62393	- 7.39485	21.1239
Present $\times 10^{-3}$	- 4.756756	9.081081	- 7.72818	16.62393	- 7.39485	21.1239

Conclusions

- Formulation of interval finite element methods (IFEM) is introduced
- EBE approach was used to avoid overestimation
- Penalty approach for IFEM
- Enclosure was obtained with few iterations
- Problem size does not affect results accuracy
- For small *stiffness* uncertainty, the accuracy does not deteriorate with the increase of *load* uncertainty
- In statically determinate case, exact hull was obtained by non-iterative approach